



Engineers Hub

For GATE, ESE & PSUs

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where Quality Matters...

GATE - 2021

Mechanical Engineering (ME)

Forenoon Session

Questions & Solutions (Memory Based)

1. * HT

Heisler chart

- Biot Number & Fourier Number

2. * ATD

Relative Humidity

$$\phi = 50\%$$

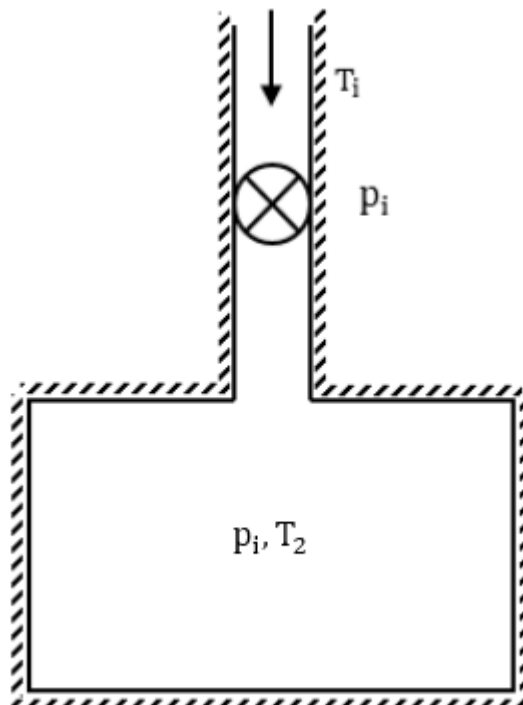
$$p_v \text{ \& } p_{vs}$$

$$\phi = \frac{p_v}{p_{vs}}$$

$$0.5 = \frac{p_v}{p_{vs}}$$

$$p_v = 0.5 p_{vs}$$

3. *ATD



$$T_i = 400 \text{ K}$$

$$\gamma = 1.4$$

$$T_2 = \gamma T_1$$

$$T_2 = 1.4 \times 400$$

$$T_2 = 560 \text{ K}$$

4. $\lim_{x \rightarrow 0} \frac{(1 - \cos x)}{x^2}$ $\left(\frac{0}{0}\right)$ form

Apply L.H Rule

$$\begin{aligned} & \lim_{x \rightarrow 0} 0 + \frac{\sin x}{2x} \\ &= \frac{1}{2} \lim_{x \rightarrow 0} \frac{\sin x}{x} \\ &= \frac{1}{2} \times 1 \\ &= \frac{1}{2} \end{aligned}$$

5. $L\{S(t - a)\} = \int_0^\infty e^{-st} \cdot \delta(t - a) dt = e^{-st}|_{t=a} \text{ (since } a > 0)$
 $= e^{-as}$



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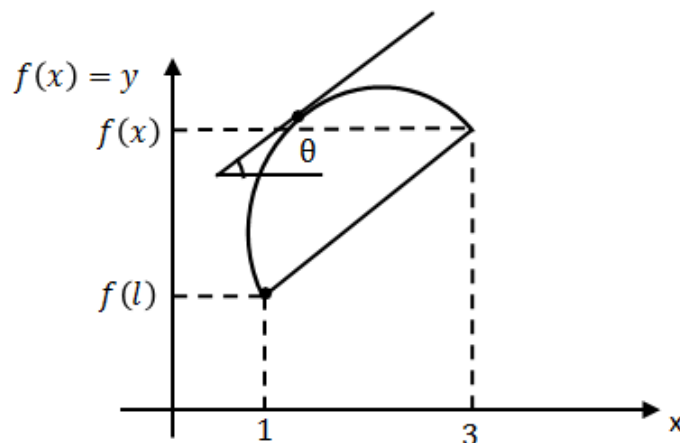
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6. $f(x) = x^2 - 2x + 2$ Continuous in $[1,3]$, the point x at which the tangent to $f(x)$ lies parallel to the line segment joining $(1, f(1))$, $(3, f(3))$

Solution:

$$f'(c) = \frac{f(b) - f(a)}{b - a}$$



$$f'(x) = 2x - 2$$

$$f'(C) = 2C - 2$$

$$f(3) = 9 - 2 \times 3 + 2 = 11 - 6 = 5$$

$$f(1) = 1 - 2 + 2 = 1$$

$$2C - 2 = \frac{5 - 1}{3 - 1}$$

$$2(C - 1) = \frac{4}{2} = 2$$

$$C = 2$$

7. *FM



No pressure gradient

$$\rho = 800 \text{ kg/m}^3$$

$$v = 1.25 \times 10^{-4} \text{ m}^2/\text{s}$$

$$\tau = ?$$

$$\tau = \mu \frac{du}{dy}$$

$$\tau = \rho v \frac{U}{Y}$$

$$\tau = 800 \times 1.25 \times 10^{-4} \times \frac{4}{5 \times 10^{-3}}$$

$$\tau = 80 \text{ N/m}^2$$

8. TD

At least one isothermal process

- * Brayton & Rankine (×)
- * Otto & Diesel (×)
- * Carnot & stirling (Right)

9. *TD

Compressibility Factor

$$Z = 0.95$$

$$p = 1500 \text{ kPa}$$

$$\bar{v} = 2.75 \text{ m}^3/\text{kmol}$$

$$T = ?$$

$$Z = \frac{PV/nT}{\bar{R}}$$

$$Z = PV/n\bar{R}T$$

$$Z = \frac{p\bar{v}}{\bar{R}T}$$

$$T = \frac{1500 \times 2.75}{8.314 \times 0.95}$$

$$= 522.26K$$

$$= 249.11^{\circ}C$$

10. $\int_C \frac{\cosh 3z}{2z}, |z| = 1$

$$\oint f(z)dz = 2\pi i[\Sigma R]$$

$\Rightarrow$ Singularity $\rightarrow z = 0$

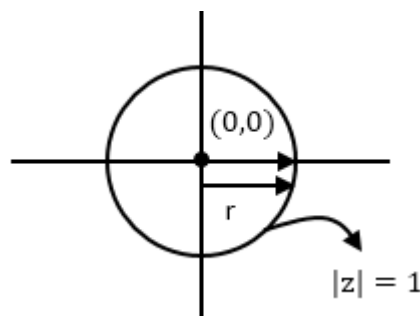
Path is $|z| = 1 \Rightarrow$

$$z = x + iy$$

$$|z| = \sqrt{x^2 + y^2}$$

$$|z| = 1$$

Equation of circle centre at origin and radius = 1



Singularity:-

$z = 0 \rightarrow$ Simple pole

$$R = \lim_{z \rightarrow a} (z - a)f(z)$$

$$\Rightarrow \lim_{z \rightarrow 0} z \times \frac{\cosh(3z)}{2z}$$

$$\Rightarrow 1/2$$

$$\oint \frac{\cosh 3z}{2z} = 2\pi i \times R = 2\pi i \times \frac{1}{2} = \pi i$$



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$$11. \sin x \frac{dy}{dx} + y \cos x = 1$$

$$y\left(\frac{\pi}{2}\right) = \frac{\pi}{2}$$

$$y\left(\frac{\pi}{6}\right) = ??$$

Solution:

$$\frac{dy}{dx} + y \cot x = \operatorname{cosec} x$$

$$\frac{dy}{dx} + py = Q$$

$$I.F = e^{\int p dx}$$

$$\Rightarrow e^{\cot x dx}$$

$$\Rightarrow e^{\log|\sin x|}$$

$$= \sin x$$

General Solution:

$$y(I.F) = \int Q(I.F) dx + C$$

$$y \sin x = \int \operatorname{cosec} x \cdot \sin x dx + C$$

$$y \sin x = \int 1 \cdot dx + C$$

$$y \sin x = x + C$$

$$\text{Given } y\left(\frac{\pi}{2}\right) = \frac{\pi}{2}$$

$$\frac{\pi}{2} \left(\sin \frac{\pi}{2} \right) = \frac{\pi}{2} + C$$

$$\frac{\pi}{2} \cdot 1 = \frac{\pi}{2} + C$$

$$C = 0$$

$$y \sin x = x + 0$$

$$y \sin x = x$$

$$y \left(\sin \frac{\pi}{6} \right) = \frac{\pi}{6}$$

$$y \left(\frac{1}{2} \right) = \frac{\pi}{6}$$

$$y = \frac{\pi}{3}$$

12.*ATD : Reheat Rankine cycle

$$w_{01} = 750 \text{ kJ/kg}$$

$$w_{02} = 1500 \text{ kJ/kg}$$

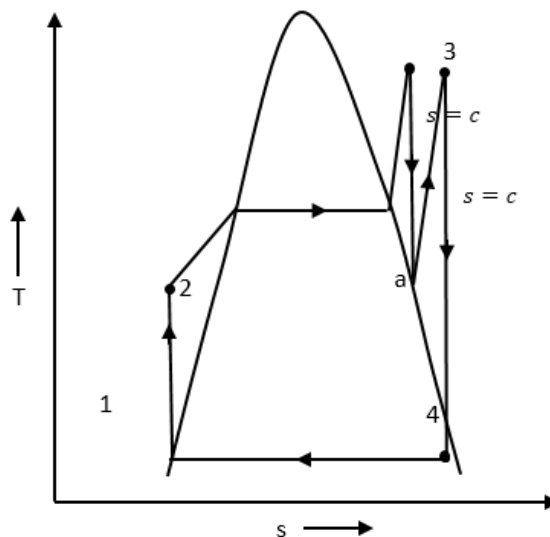
$$\eta = 50\%$$

$$(h_f)_{P_c} = 200 \text{ kJ/kg}$$

$$(h_g)_{P_c} = 2600 \text{ kJ/kg}$$

$$w_i = 20 \text{ kJ/kg}$$

$$x_4 = ?$$



$$\eta_{RR} = \frac{\Sigma w}{q_s}$$

$$0.5 = \frac{750+1500-20}{q_s}$$

$$q_s = \frac{2230}{0.5} = 4460 \text{ kJ/kg}$$

$$\Sigma q = \Sigma w$$

$$q_s - q_r = 2230$$

$$4460 - q_r = 2230$$

$$q_r = 2230 \text{ kJ/kg}$$

$$x_4(h_{fg})_c = 2230 \text{ kJ/kg}$$

$$x_4 = \frac{2230}{2600 - 200}$$

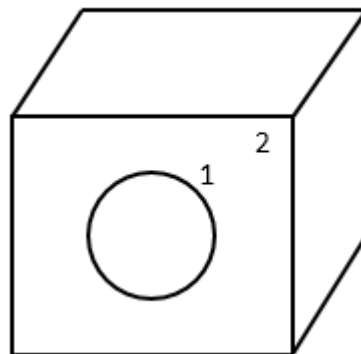
$$x_4 = \frac{2230}{2400}$$

$$x_4 = 0.929$$

$$x_4 = 92.9\%$$

$$\text{Ans: } 93\%$$

13. Shape factor:



$$L = 30 \text{ mm}$$

$$r = 10 \text{ mm}$$

$$F_{22} + F_{21} = 1$$

$$F_{11} + F_{12} = 1$$

$$F_{11} + F_{12} = 1$$

$$F_{12} = 1$$

$$A_1 F_{12} = A_2 F_{21}$$

$$F_{21} = \frac{A_1}{A_2} \text{ (Since } F_{12} = 1)$$

$$F_{21} = \frac{4\pi r^2}{6a^2}$$

$$F_{21} = 0.2327$$

$$F_{22} = 1 - F_{21}$$

$$= 1 - 0.2327$$

$$F_{22} = 0.7673$$

$$F_{22} = 0.77$$

14. * HT (Fin)

$$\dot{Q}_{\text{fin}} = \sqrt{hpkA_c} \theta_b$$

$$\theta_b = T_b - T_{\infty}$$

$$\dot{Q}_{\text{fin}} \propto \sqrt{k}$$

$$k_2 = 2k_1$$

$$\frac{\dot{Q}_2}{\dot{Q}_1} = \sqrt{\frac{k_2}{k_1}} = \sqrt{\frac{2k_1}{k_1}}$$

$$\frac{\dot{Q}_2}{\dot{Q}_1} = \sqrt{2}$$

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15. * TD

$$z = f(x, y)$$

$$dz = Mdx + Ndy$$

$$dU = Tds + \tau dL$$

$$\left. \frac{\partial T}{\partial L} \right|_s = \left. \frac{\partial \tau}{\partial s} \right|_L$$

16. * HM

Simple Pelton Turbine

$$N = 300 \text{ RPM}, D = 2 \text{ m}, v_1 = 40 \frac{\text{m}}{\text{s}} = V_{w1}$$

$$Q = 5 \text{ m}^3/\text{s}$$

$$\alpha = 165^\circ$$

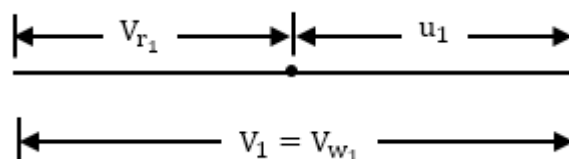
Power output = (MW)

$$\text{Power} = \dot{m} [\bar{V}_{w1} - \bar{V}_{w2}] x^4$$

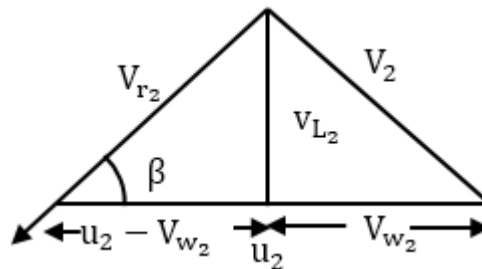
$$\dot{m} = \rho Q, u_1 = u_2 = u = \frac{\pi DN}{60}$$

$$u = 31.416 \text{ m/s}$$

Inlet vel. Diagram



$$V_{r1} = 8.584 \text{ m/s}$$



$$\beta_2 = 180 - 165 = 15^\circ$$

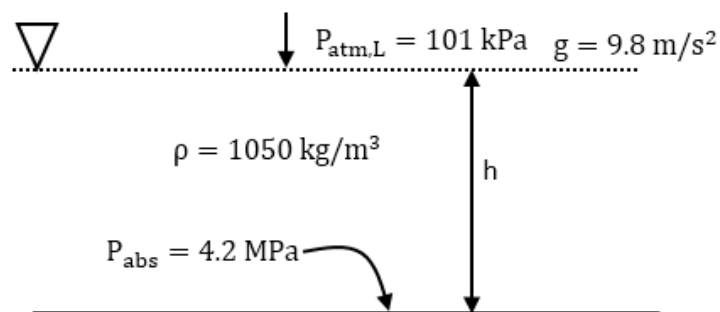
$$\cos \beta_2 = \frac{u_2 - V_{w2}}{V_{r2}}$$

$$V_{r2} = V_{r1} \times k$$

$$k = 1$$

$$\text{Power} = 2.6808 \text{ MW}$$

17.* FM



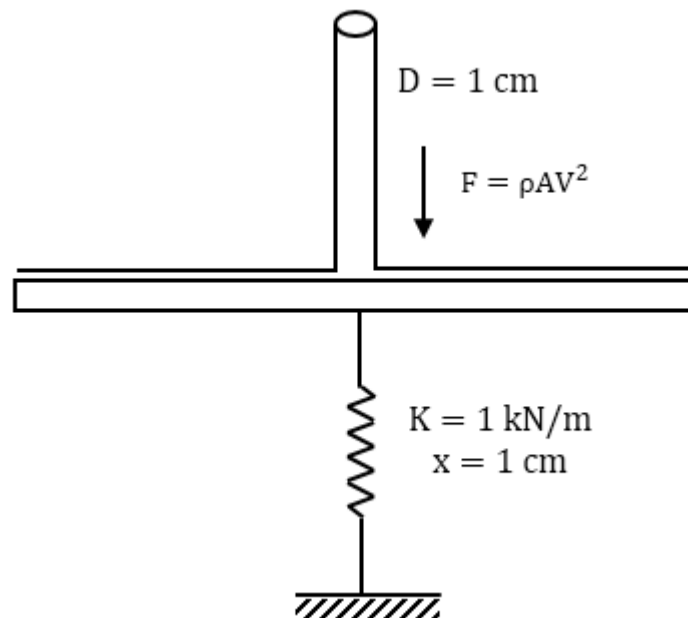
$$P_{ab} = P_{atm,L} + \rho gh$$

$$h = \frac{P_{abs} - P_{atm,L}}{\rho g}$$

$$= \frac{4200 - 101}{1.050 \times 9.8}$$

$$h = 398.34 \text{ m}$$

18. * FM



Water

$$\rho AV^2 = Kx$$

$$V^2 = \frac{Kx}{\rho A}$$

$$V^2 = \frac{10^3 \times 10^{-2}}{10^3 \times \frac{\pi}{4} \times 10^{-4}}$$

$$V^2 = \frac{400}{\pi}$$

$$V = \frac{20}{\sqrt{\pi}}$$

$$V = 11.28 \text{ m/s}$$



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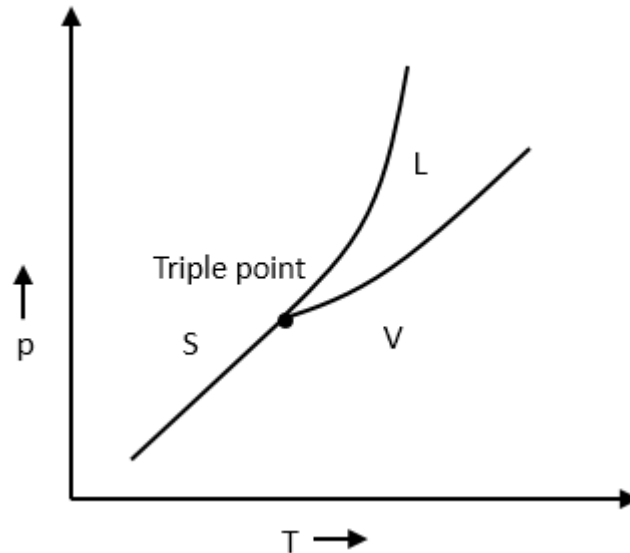
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19. * NH_3



L – V:

$$\ln P = 24.38 - \frac{3063}{T}$$

S → V:

$$\ln P = 27.92 - \frac{3754}{T}$$

$$\Rightarrow 24.38 - \frac{3063}{T} = 27.92 - \frac{3754}{T}$$

$$T = 195 \text{ K}$$

20. Step size = h

$$\frac{y_{i+1} - y_i}{h} = -\pi y_i \rightarrow \text{Solution is stable}$$

⇒ Forward Euler Method

$$\frac{dy}{dx} = \lambda y$$

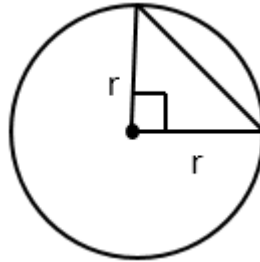
For stability of solution

$$0 \leq h \leq \frac{2}{|\lambda|}$$

$$\lambda = -\pi$$

$$0 \leq h \leq \frac{2}{\pi}$$

21.



Area of triangle = 50 cm^2

Area of the circle=??

$$\frac{1}{2} \times r \times r = 50$$

$$r^2 = 100$$

$$r = 10$$

Area of the circle

$$\pi \times r^2 \Rightarrow \pi \times (10)^2$$

$$= 100\pi$$

22.

$$t = 1.55 \text{ mm}$$

$$I = 10000 \text{ A}$$

$$\text{time} = 0.25 \text{ sec}$$

$$V_n = 70 \text{ mm}^3$$

$$E/\text{vol} = 12 \text{ J/mm}^3$$

$$R = 0.0001 \Omega$$

$$\eta_{\text{th}} = \frac{\mu_{\text{req}}}{\mu_{\text{sup}}}$$

$$70 \times 12 = 840$$

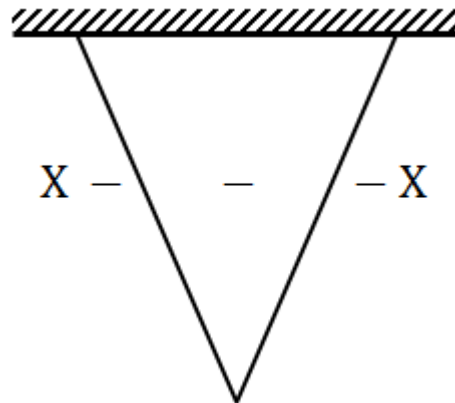
$$\eta_{th} = 0.336$$

$$I^2 R t = 10000^2 \times 0.0001 \times 0.25 \\ = 2500 \text{ J}$$

$$\eta_{th} = 0.336 \times 100 = 33.6\%$$

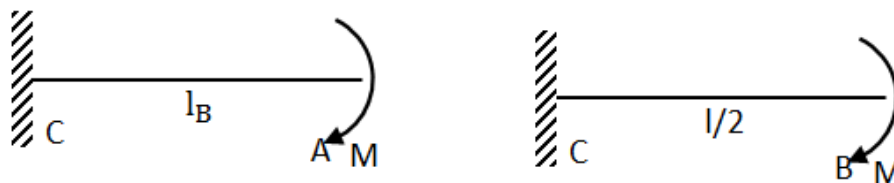
23.

Find deflection at mid height due to self-weight.



$$S_{(x-x)} = \frac{\gamma l^2}{6E} - \frac{\gamma (l/2)^2}{6E} = \frac{\gamma l^2}{6E} \left(1 - \frac{1}{4}\right) = \frac{\gamma l^2}{8E} = \frac{\gamma l^2}{8E}$$

24.



Deflection at mid of beam = ?

$$y_B = \frac{M(l/2)^2}{2EI} = \frac{Ml^2}{8EI}$$



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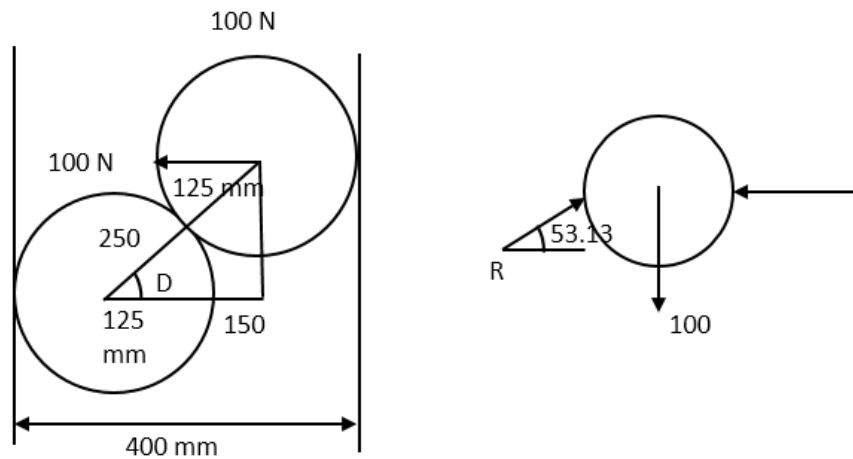
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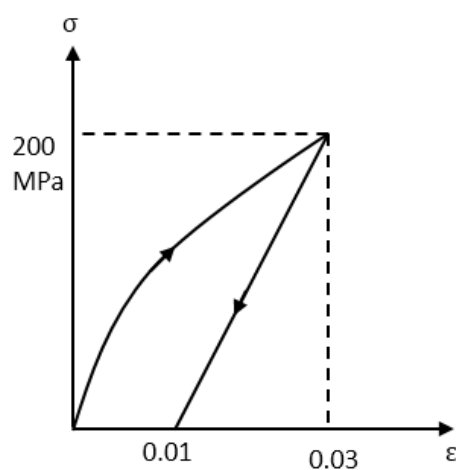
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25. Reaction between spheres?



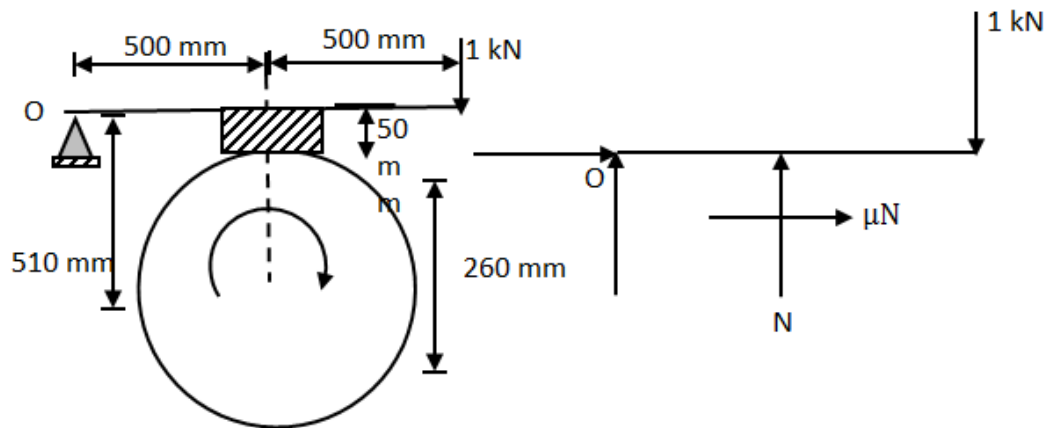
$$\begin{aligned}\Sigma F_y &= 0 \\ R \sin 53.13 &= 100 \\ R &= 125 \text{ N}\end{aligned}$$

26.



$$\begin{aligned}\epsilon_{\text{elastic}} &= 0.03 - 0.01 \\ &= 0.02 \\ \epsilon_{\text{plastic}} &= 0.01\end{aligned}$$

27. Breaking Torque for below shown case, when $\mu = 0.4$ is _____.



$$\mu = 0.4$$

$$T_B = ?$$

$$\sum M_O = 0$$

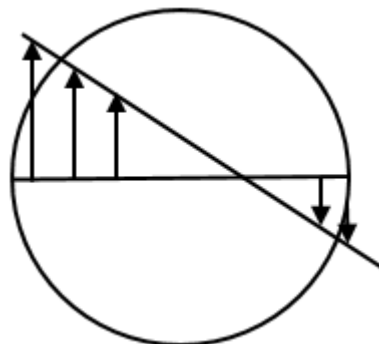
$$N \times 500 + 0.4N \times 50 = 1000 \times 1000$$

$$N = 1523.07$$

$$T_B = \mu N r$$

$$= 0.4 \times 1523.07 \times 0.260 = 158.7 \text{ N-m}$$

28.



Stress distribution in cross-section of helical spring

- (A) only torsional stress
- (B) only direct shear stress
- (C) torsional + direct shear stress

(D) torsional + direct shear stress + curvature effects.

Answer: C

29.

$$\lambda = 12/\text{hr}$$

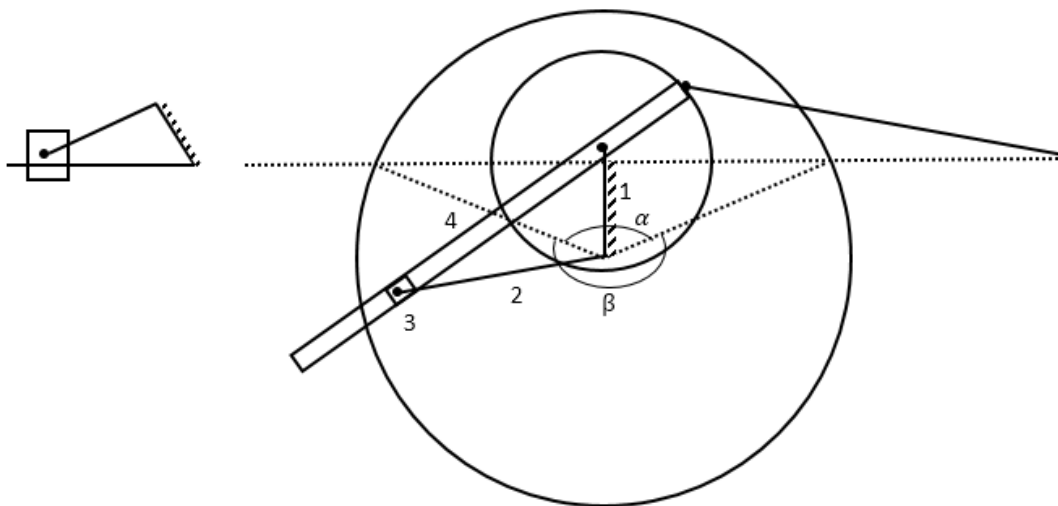
$$\mu = \text{mean} (4 \text{ mm}) = \frac{60}{4} = 15/\text{hr}$$

$$f = \frac{\lambda}{\mu}$$

$$= \frac{12}{15}$$

$$L_s = \frac{f}{1-f} = \frac{0.8}{1-0.8} = 4$$

30. Whitworth quick return ratio =?



$$\text{QRR} = \frac{\beta}{\alpha}$$

$$= 2$$



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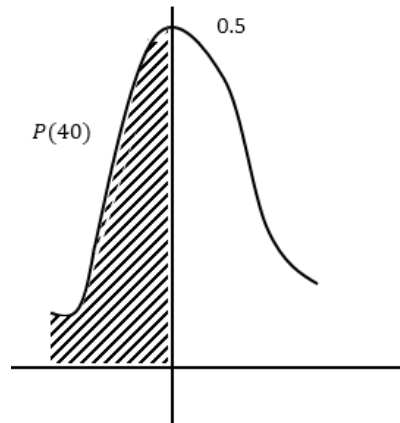
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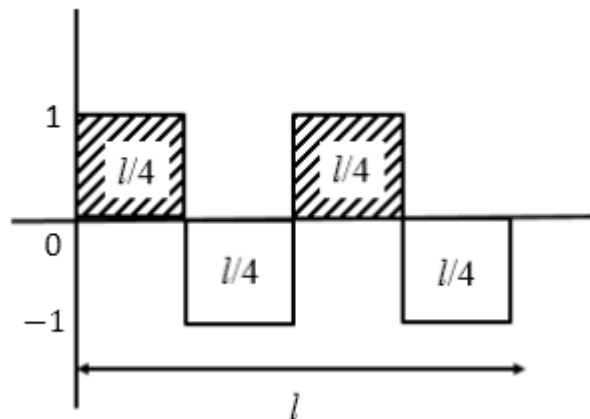
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31.



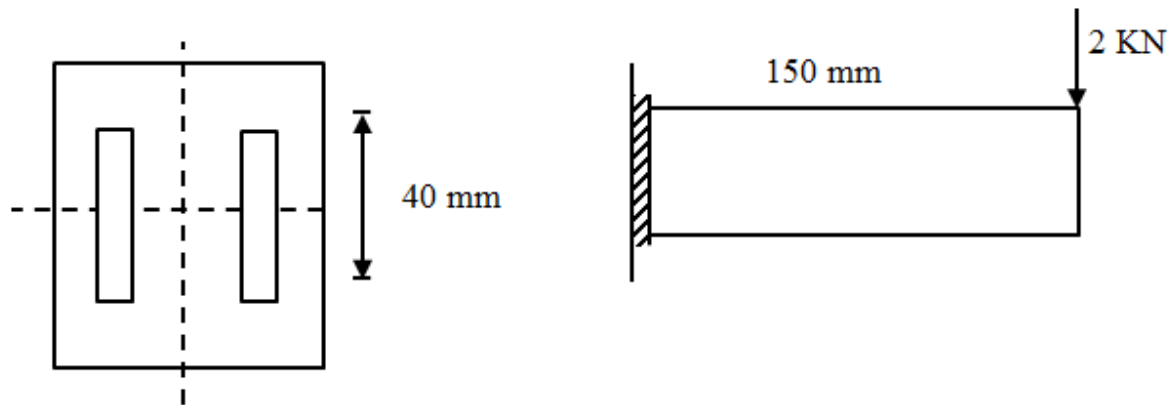
	A	B	C	D
Duration	6	11	8	15 = 40 p (40)
V	4	9	4	9

32.



$$\text{CEA} = \frac{\frac{1}{4} + \frac{1}{4} + \frac{1}{4} + \frac{1}{4}}{1} = 1$$

33.



$$\tau_{\text{allow}} = 60 \text{ MPa}$$

Leg of weld = ?

$$\tau = \frac{2000}{2 \times 40t} = \frac{25}{t} \text{ MPa}$$

$$\sigma_b = \frac{m \cdot y_{\text{max}}}{f} = \frac{(2000 \times 150) \times (40/2)}{2 \times \frac{t \times 40^3}{12}} = \frac{562.5}{t}$$

$$\tau_{\text{max}} = \sqrt{\left(\frac{\sigma_b}{2}\right)^2 + \tau^2}$$

$$60 = \frac{1}{t} \sqrt{\left(\frac{562.5}{2}\right)^2 + 25^2}$$

$$t = 4.706 \text{ mm}$$

$$0.707 h = 4.706 \text{ mm}$$

$$h = 6.656 \text{ mm}$$

34.

$$S_{ut} = 600 \text{ MPa}$$

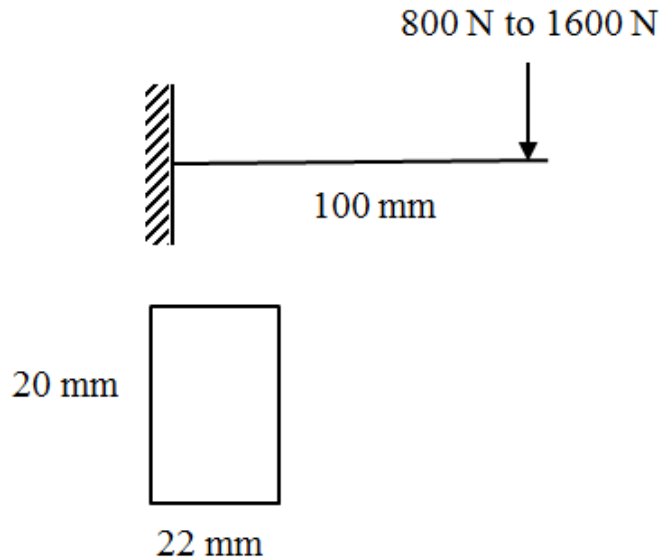
$$S_{yt} = 500 \text{ MPa}$$

$$S_z = 200 \text{ MPa}$$

Find FOS by

Modified Goodman's

Criteria



$$\sigma_{\max} = \frac{(1600 \times 100)}{\left(\frac{12 \times 20^3}{6}\right)} = 200 \text{ MPa}$$

$$\sigma_{\min} = \frac{(800 \times 100)}{\left(\frac{12 \times 20^3}{6}\right)} = 100 \text{ MPa}$$

$$\sigma_m = \frac{200 + 100}{2} = 150 \text{ MPa}, \quad \sigma_g = \frac{200 - 100}{2} = 50 \text{ MPa}$$

Goodman's

$$\frac{\sigma_m}{S_{yt}} + \frac{\sigma_a}{S_e} = \frac{1}{\text{FOS}}$$

$$\frac{150}{600} + \frac{50}{200} = \frac{1}{\text{FOS}}$$

$$\text{FOS} = 2$$

$$\text{FOS} \leq 2.5$$

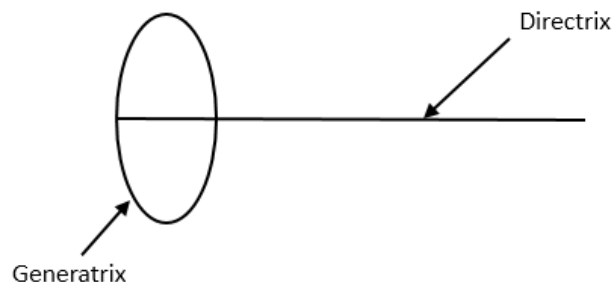
Langer's

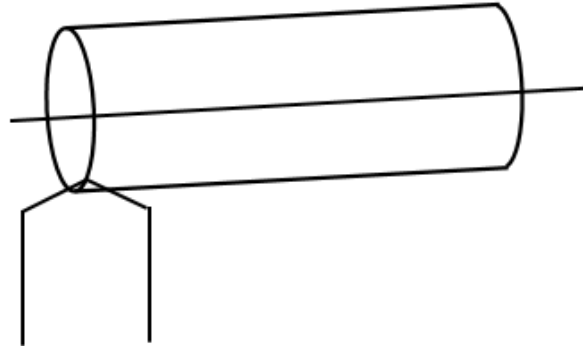
$$\sigma_m + \sigma_a = \frac{S_{yt}}{\text{FOS}}$$

$$150 + 50 = \frac{500}{\text{FOS}}$$

$$\text{FOS} = 2.5$$

35.





A. Cylinder

36.

Finishing Job (Hole)



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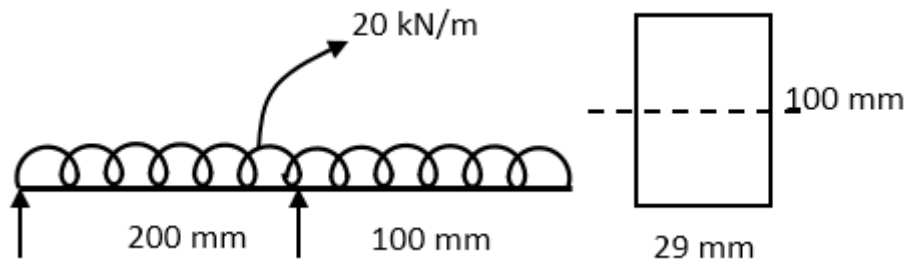
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37.



$$(\sigma_b)_{\max} = ?$$

$$\frac{M}{I} = \frac{\sigma_0}{y}$$

$$\sigma_b = \frac{MY}{I} = \frac{M}{\left(\frac{I}{Y}\right)} = \frac{M}{Z}$$

$$Z = \frac{bd^2}{6}$$

$$Z = \frac{24 \times (100)^2}{6}$$

$$Z = 4 \times (10)^4$$

$$(\sigma_m)_{\max} = \frac{M}{Z} = \frac{10 \text{ kN-m}}{4 \times (10)^4 (\text{mm})^3}$$

$$= \frac{10 \times 1000 \times 1000}{4 \times (10)^4 \text{ mm}^3} \text{ N-mm}$$

$$= 250 \text{ N/mm}^2$$

38.

	A	B	C	D	E	F	G	H
M-1	2	4	8	9	7	6	5	10
M-2	6	1	3	7	9	5	2	4

A	E	D	F	H	C	G	B
---	---	---	---	---	---	---	---

39.

Machining

$$\alpha = 12$$

$$F_C = 1000 \text{ N}$$

$$F = 600 \text{ N}$$

$$t_1 = 0.75 \text{ mm}$$

$$t_2 = 15 \text{ mm}$$

$$F_s = ?$$

$$F_s = R \cos(\phi + \lambda - \alpha)$$

$$\tan \phi = \frac{r \cos \alpha}{1 - r \sin \alpha}$$

$$r = 0.5$$

$$\phi = \tan^{-1} \left(\frac{0.5 \cos 12}{1 - 0.5 \times \sin 12} \right)$$

$$\phi = 28.62$$

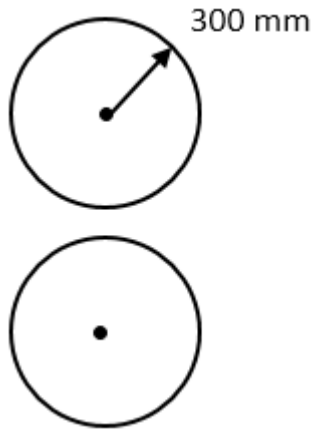
$$2\phi + \lambda - \alpha = \frac{\pi}{2}$$

$$\lambda = 44.76$$

$$R = 1848.01 \text{ N}$$

$$F_s = R \cos(\phi + \lambda - \alpha) = 885.19 \text{ N}$$

40.



20 mm

18 mm

h_1

h_2

$k = 300 \text{ MPa}$

$n = 0.2$

$\sigma_f = 300 e^{0.2}$

$F = \bar{\sigma}_f \times W \times L_p$

$\Rightarrow F = \frac{k}{n+1} e_{\max}^n \times W \times \sqrt{R\Delta h}$

$\Rightarrow F = \frac{k}{n+1} \ln\left(\frac{h_2}{h_1}\right) \times W \times \sqrt{R\Delta h}$

$k = 300, (n+1) = 1.2, h_2 = 18, h_1 = 20$

$R = 300, h = 2$

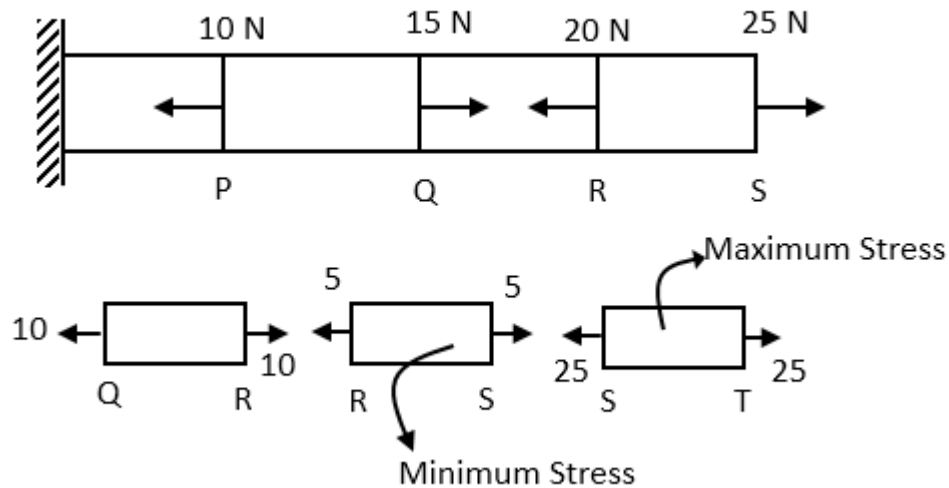
$F = 955.92 \text{ kN}$

41.

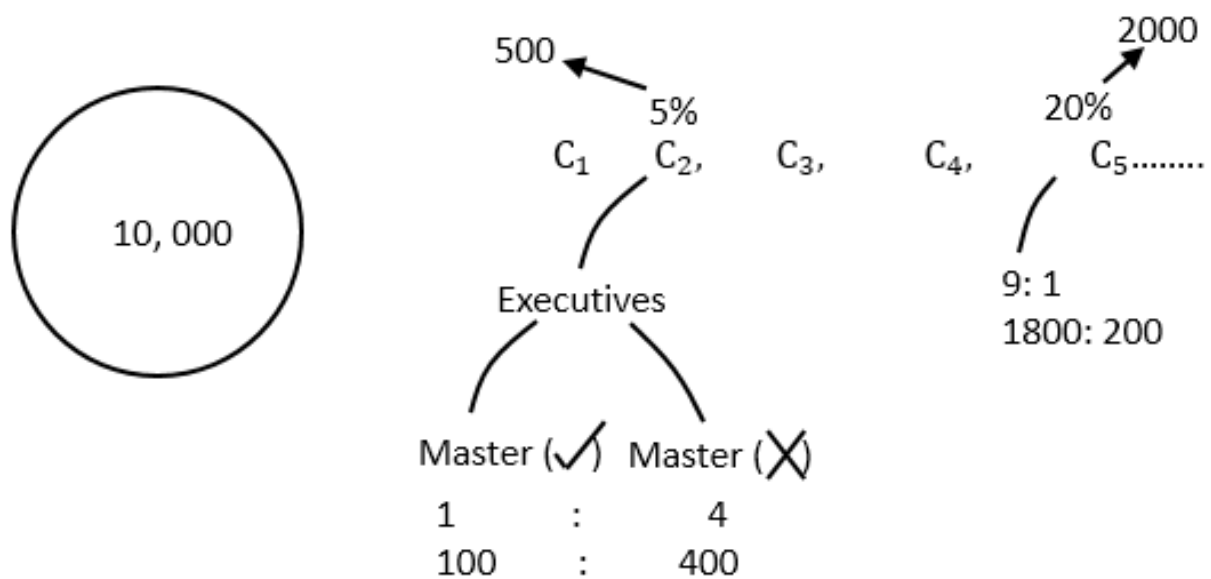
Area length

Max stress = ?

Min stress = ?



42.



$$1800 + 100 = 1900$$

43.

P, Q, R, S, T

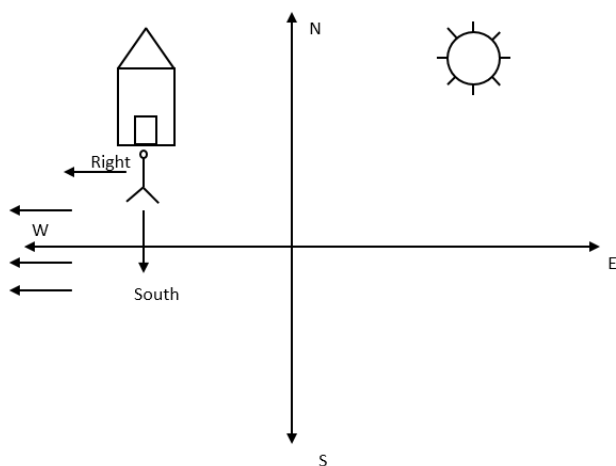
Q	P/T	R	P/T	S	}	4
Q	P/T	R	S	P/T		
R	T/P	Q	P/T	S	}	2
T/P	Q	P/T	R	S		
S	R	P/T	Q	T/P	}	2
S	P/T	Q	T/P	R		
P/T	S	R	T/P	Q	}	4
S	T/P	R	P/T	Q		

Answer is 16

44.

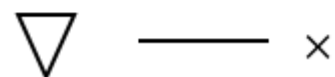
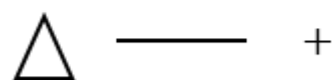
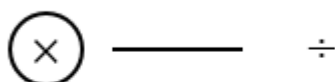
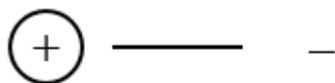
Farm P			Farm Q
Hen	Ducks	Goats	416
13 (65	91	169)	
			Coprimes
5:	7:	13	5: 14: 13
			10: 21: 26

45.



South facing.

46.



$$\triangle^2 \oplus^3 \triangle (4 \otimes^2) \nabla 4$$

$$+2 - 3 + (4 \div 2 \times 4)$$

$$-1 + (8)$$

$$= 7$$